

DESIGN AND ANALYSIS OF THERMAL BARRIER COATING ON GASTURBINE BLADE

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Abstract: Thermal barrier coatings (TBCs) are deposited on the turbine blade to reduce the temperature of underlying substrate, as well as providing protection against the oxidation and hot corrosion from high temperature gas. Optimal ceramic top-coat thickness distribution on the blade can improve the performance and efficiency of the coatings. Design of the coatings thickness is a multi-objective optimization problem due to the conflicts among objectives of high thermal insulation performance, long operation durability, and low fabrication cost. This work developed a procedure for designing the TBCs thickness distribution of 100µm to 500µm for the gas turbine blade. The base material of blade geometry is created using Nickel alloy and its coating material is selected as partially stabilized zirconia. Three-dimensional finite element models were built using CATIA and analyzed by ANSYS WORKBENCH, and weighted-sum approach was employed to solve the multi objective optimization problem herein. Suitable multi region top-coat thickness distribution scheme was designed with the considerations of manufacturing accuracy, productivity, and fabrication cost.

Keywords: Thermal Barrier Coatings; Oxidation; corrosion; Ceramic Top-coat thickness; Durability; Low fabrication cost

1. INTRODUCTION

The objective of this project is to design and stresses analyze a turbine blade of a jet engine. An investigation for the usage of new materials is required. In the present work turbine blade was designed with two different materials named as Inconel 718 and Titanium T-6. An attempt has been made to investigate the effect of temperature and induced stresses on the turbine blade. A thermal analysis has been carried out to investigate the direction of the temperature flow which is been develops due to the thermal loading. A structural analysis has been carried out to investigate the stresses, shear stress and displacements of the turbine blade which is been develop due to the coupling effect of thermal and centrifugal loads. An attempt is also made to suggest the best material for a turbine blade by comparing the results obtained for two different materials (Inconel 718 and titanium T6). Based on the plots and results Inconel718 can be consider as the best material which is economical, as well as it has good material properties at higher temperature as compare to that of TitaniumT6 [1-5]. In the present work the first stage rotor blade of a two-stage gas turbine has been analyzed for static structural, steady state thermal, modal and high cycle fatigue using ANSYS 17. An attempt has been made to investigate the effect of temperature and induced stresses on the turbine blade. A structural analysis has been carried out to investigate the stresses and displacements of the turbine blade which is been develop due to the coupling effect of thermal and centrifugal loads. A steady state thermal analysis has been carried out to investigate the direction of the temperature flow which is been develops due to the thermal loading. An attempt is also made to suggest the best material for a turbine blade by comparing the results obtained for three different materials such as Titanium Ti 6Al 4V, INCONEL 625 and N-155 that has been considered for the analysis. The turbine blade along with the fir tree joint is considered for the static structural, fatigue, thermal and modal analysis. The blade is modeled with CATIA V5. The geometric model of the blade profile is generated with splines and extruded to get a solid model. [5-10]. Gas turbine play a vital role in the today's industrialized society, and as the demand for power increase, the power output and thermal efficiency of gas turbine must also increase. One method of increasing both the power output and thermal efficiency of the engine is to increase the temperature of the gas entering the turbine. In the advanced gas turbine, the inlet temperature of around 1500°C is used, however, this temperature exceeds the melting temperature of the metal airfoils. Therefore, along with high temperature material development, a refined cooling system must be developed for continuous safe operation of gas turbines with high performance. Gas turbine blades are cool internally and externally. In film cooling, relatively cool air is injected from the inside of the blade which travels through the entire blade length and form a protective film around the blade-surface. In present work attempt has

been made to analyze the failure of the gas turbine blade through structural analysis. The analysis is conducted for two different blade configurations one is the base line configuration with film cooling cylindrical holes along the entire length and in another configuration is the holes along the leading edge are branched together for anti-vortex considerations. The two configurations is further study for two different pitches to diameter ratio(y/d) of the cooling holes [11-17]

3. MODELING OF GAS TURBINE BLADE

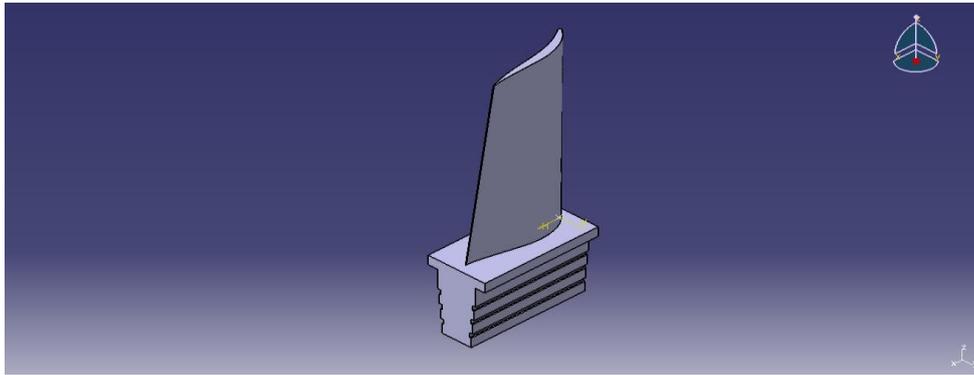


Figure 1. The gas turbine blade with 100µm coating layer

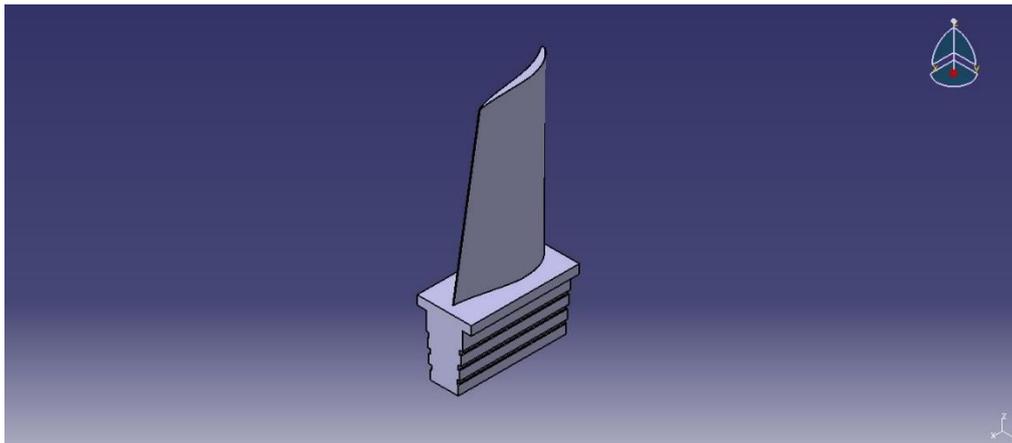


Figure 2. The gas turbine blade with 200µm coating layer

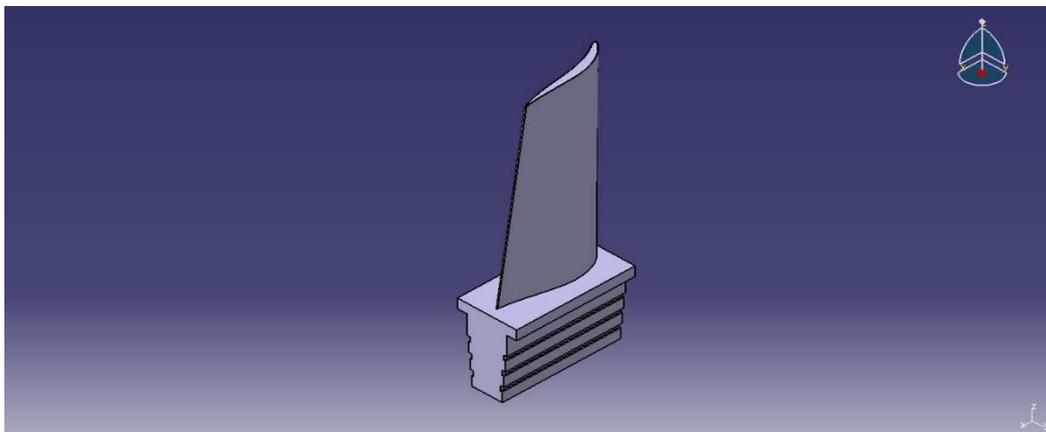


Figure 3. The gas turbine blade with 300µm coating layer

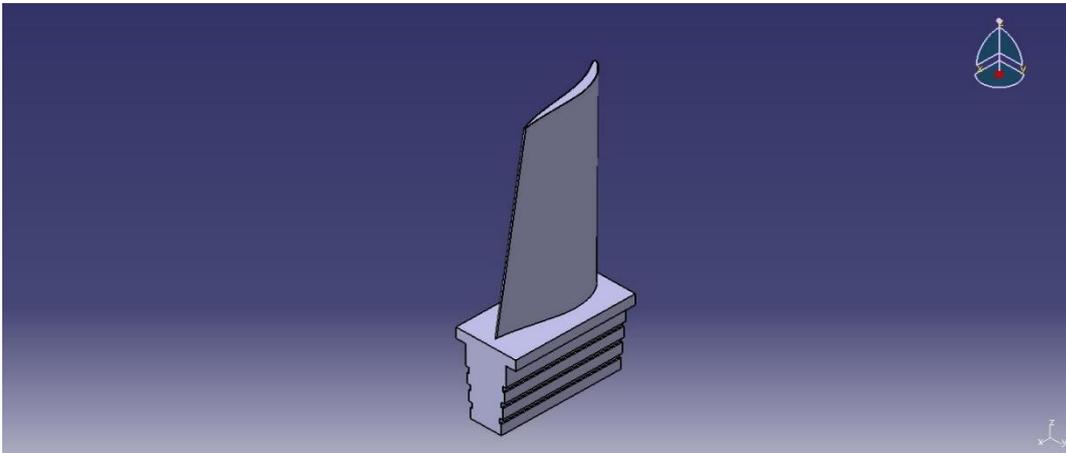


Figure 4. The gas turbine blade with 400µm coating layer

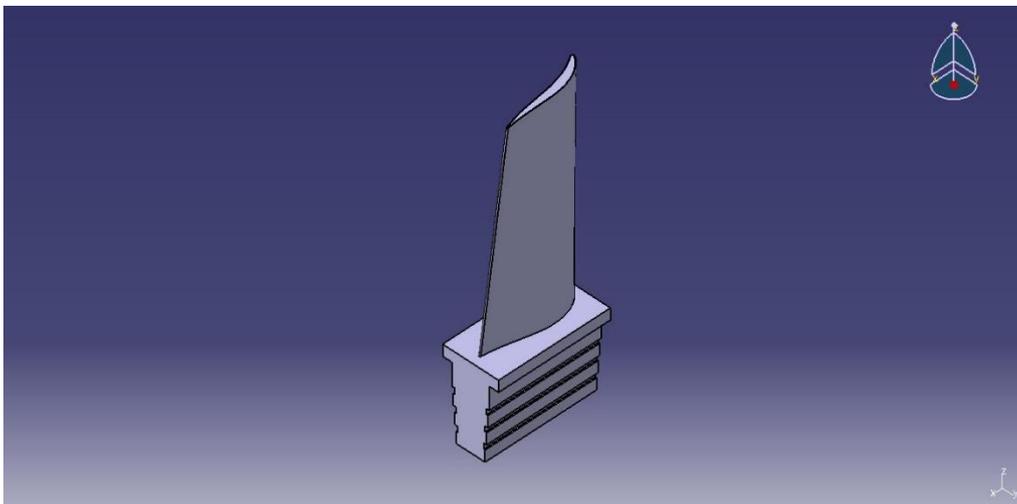


Figure 5 The gas turbine blade with 500µm coating layer

4.RESULT AND ANALYSIS

THERMAL ANALYSIS

The ANSYS/ Multi physics, ANSYS/Mechanical, ANSYS/FLOTRAN, and ANSYS/Thermal products support steady-state thermal analysis. A steady-state thermal analysis calculates the effects of steady thermal loads on a system or component. Engineer/analysts often perform a steady-state analysis before doing a transient thermal analysis, to help establish initial conditions. A steady-state analysis also can be the last step of a transient thermal analysis, performed after all transient effects have diminished.

The steady-state thermal analysis is used to determine temperatures, thermal gradients, heat flow rates, and heat fluxes in an object that are caused by thermal loads that do not vary over time. Such loads include the following:

- Convections
- Radiation
- Heat flow rates
- Heat fluxes (heat flow per unit area)
- Heat generation rates (heat flow per unit volume)
- Constant temperature boundaries



ANALYZING PROCEDURE

Step1

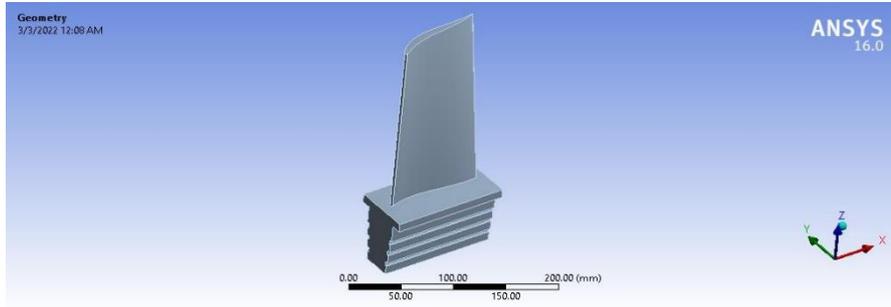


Figure 6. The model of gas turbine blade is imported in ansys workbench

Step2

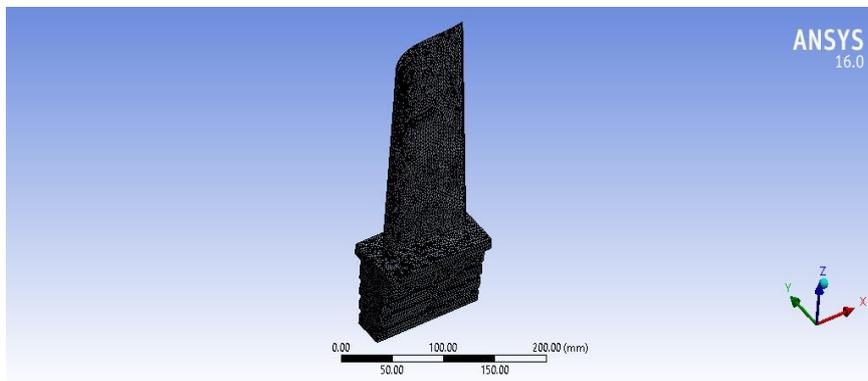


Figure 7. The meshed view of gas turbine blade

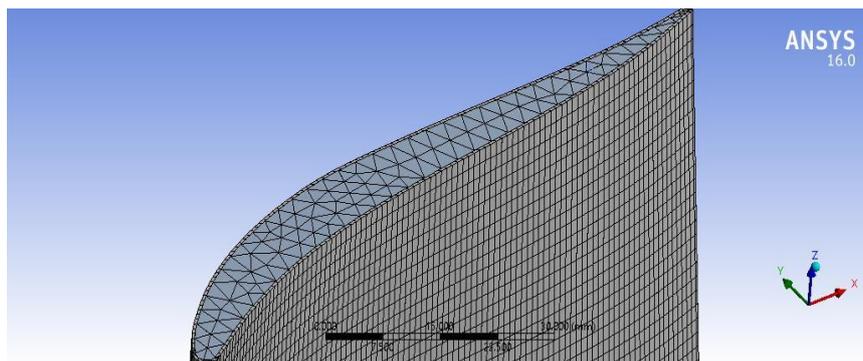


Figure 8. Details of meshes

Nodes	379113
Elements	174041

Figure 9. Quantity of mesh created



Step3

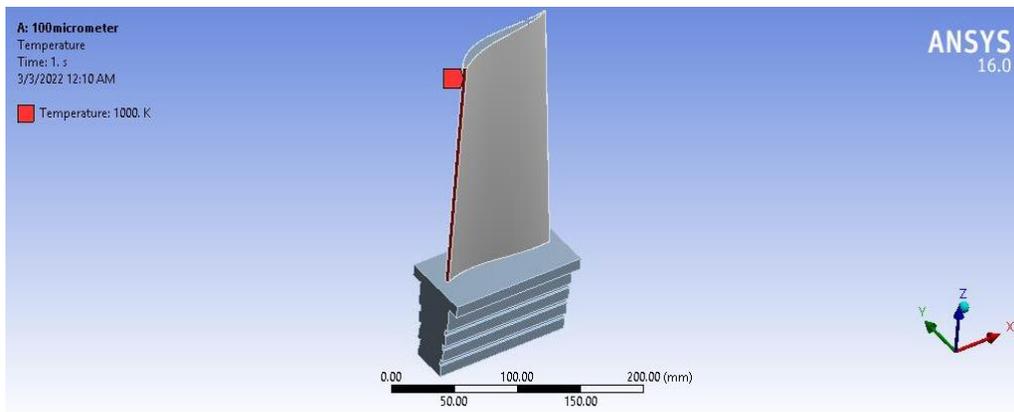
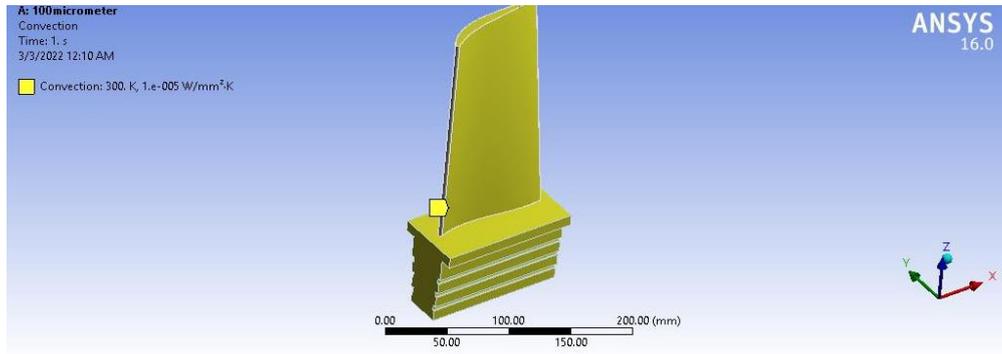


Figure 10. The temperature is applied at the leading edge of airfoil

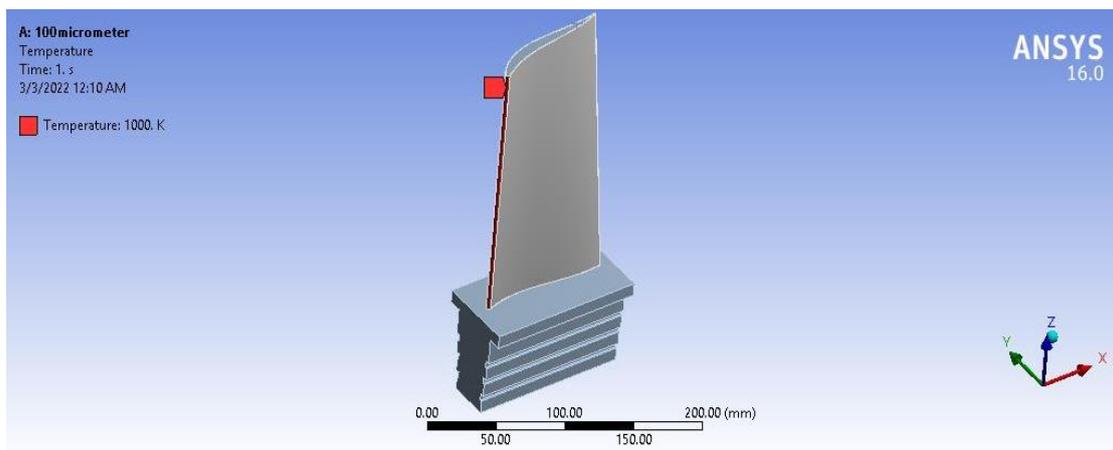


Figure 11. The convection is provided at the remaining faces of turbine blades to calculate heat transfer

Step5

Base metal- Inconel 718

Properties of Inconel 718

Inconel 718 is a nickel-based super alloy that is well suited for applications requiring high strength in temperature ranges from cryogenic up to 1400°F. Inconel 718 also exhibits excellent tensile and impact strength.

Chemical Composition of Inconel 718

The Chemical Composition of Inconel 718 are Carbon 0.08 max, Manganese 0.35 max, Phosphorus 0.015 max, Sulfur 0.015 max, Silicon 0.35 max, Chromium 17-21, Nickel 50-55, Molybdenum 2.80-3.30, Columbium 4.75-5.50, Titanium 0.65-1.15, Aluminum 0.20-0.80, Cobalt 1.00 max, Boron 0.006 max, Copper 0.30 max, Tantalum 0.05 max, Iron Balance

Resistance to Corrosion and Oxidation of Inconel 718

Inconel 718 has good resistance to oxidation and corrosion at temperatures in the alloy’s useful strength range in atmospheres encountered in jet engines and gas turbine operations.

5.4. COATING MATERIAL

International Syalons supply Zirconia 3Y-PSZ ceramics, suitable for applications in many different testing environments. These ceramics are the toughest monolithic ceramic and was developed to compliment International Syalons range of components for the metal forming industries.

Zirconia 3Y-PSZ is a partially stabilised zirconia characterised by high strength and toughness giving it excellent wear resistance as well as excellent corrosion resistance.

Physical and Mechanical Properties of Partially Stabilised Zirconia (3Y-PSZ)

Properties	Value	Units
3 point RT Modulus of Rupture (Specimen 3 x 3 x 51 span 19.05mm)	1000	MPa
Weibull Modulus	15	–
RT Unit Tensile Strength	500	MPa
RT Compressive Strength	>2000	MPa
RT Young’s Modulus of Elasticity	205	GPa
RT Hardness (HRA)	91	–
RT Hardness (Vickers Hv0.3)	1350	Kg/mm ²
Fracture Toughness K ¹ C	9.0	MPam ^{1/2}
Density	6.03	g/cc

Porosity	0	%
Thermal Expansion Coefficient (0-1200°C)	10.0×10^{-6}	K^{-1}
RT Thermal Conductivity	2.0	W/m/k
Thermal Shock Resistance	250	$\Delta T^{\circ}C$ quenched in water
Maximum Use Temperature	1000	$^{\circ}C$
RT Electrical Resistivity	109	ohm m

Table 5.4. The table below lists typical mechanical, thermal and electrical property data for Zirconia 3Y-PSZ.

Typical physical property data obtained under test conditions. All properties have been measured by independent testing authorities. The values given only apply to the test bodies on which they were determined, and therefore can only be recommended values.

5.5. RESULTS OF VARIOUS THICKNESS TBC

5.5.1. Results of thermal barrier coating of 100µm

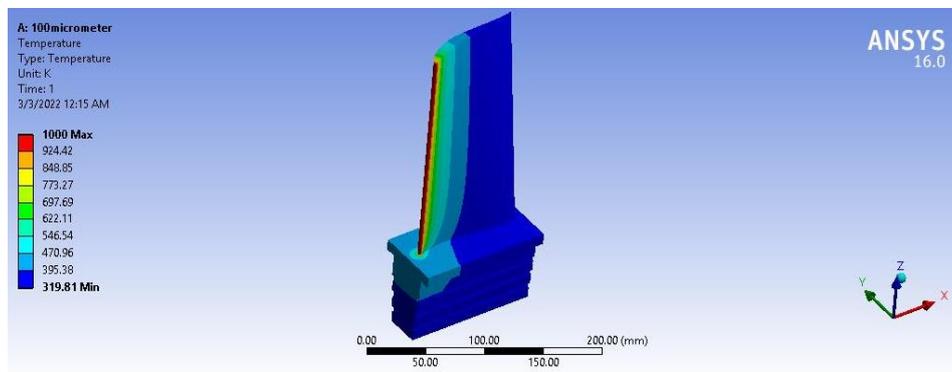


Figure 5.7.a. Temperature distribution

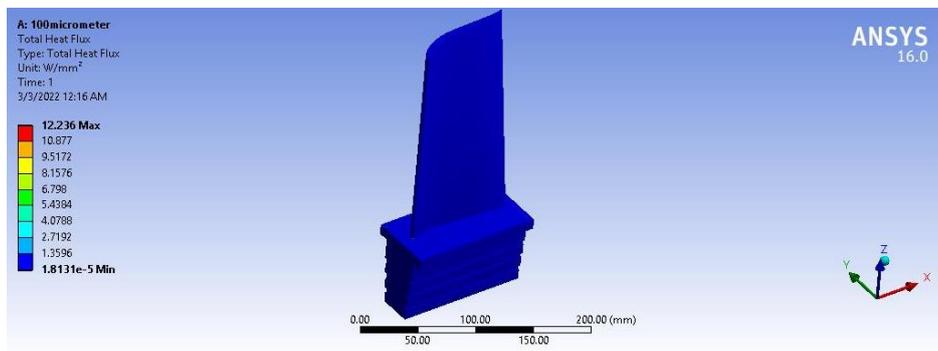


Figure 5.7.b. Heat flux experienced by the turbine blade structure

5.5.2. Results of thermal barrier coating of 200 μ m

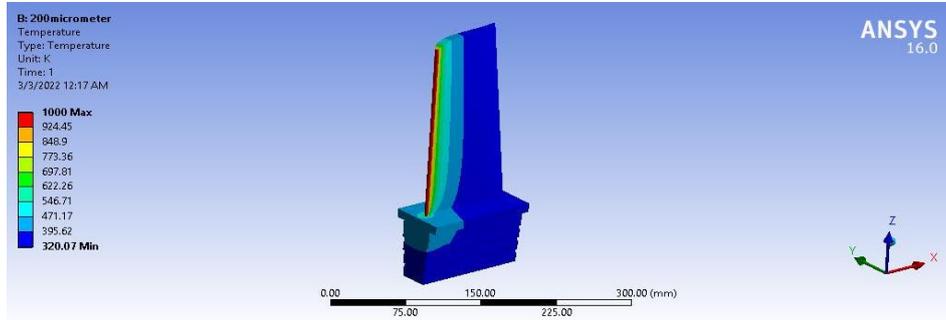


Figure 5.8.a. Temperature distribution

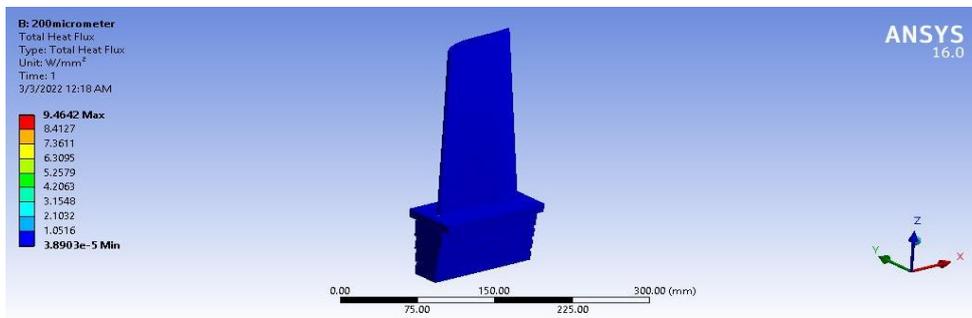


Figure 5.8.b. Heat flux experienced by the turbine blade structure

5.5.3. Results of thermal barrier coating of 300 μ m

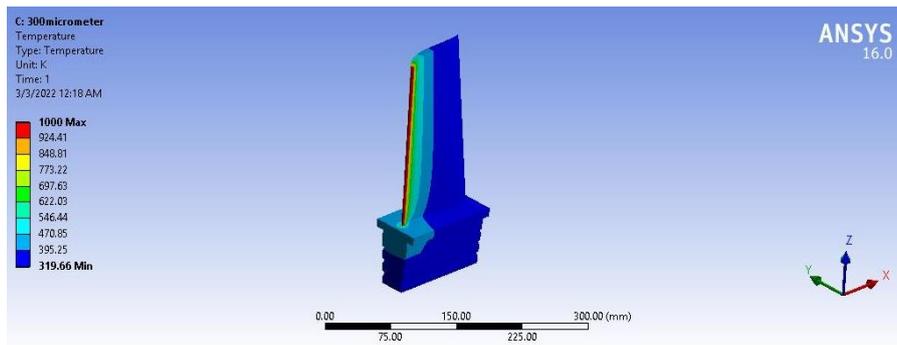


Figure 5.9.a. Temperature distribution

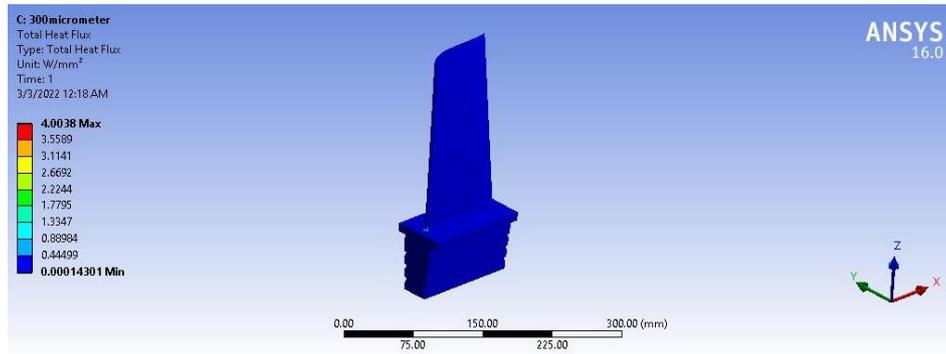


Figure 5.9.b. Heat flux experienced by the turbine blade structure

5.5.4. Results of thermal barrier coating of 400 μ m

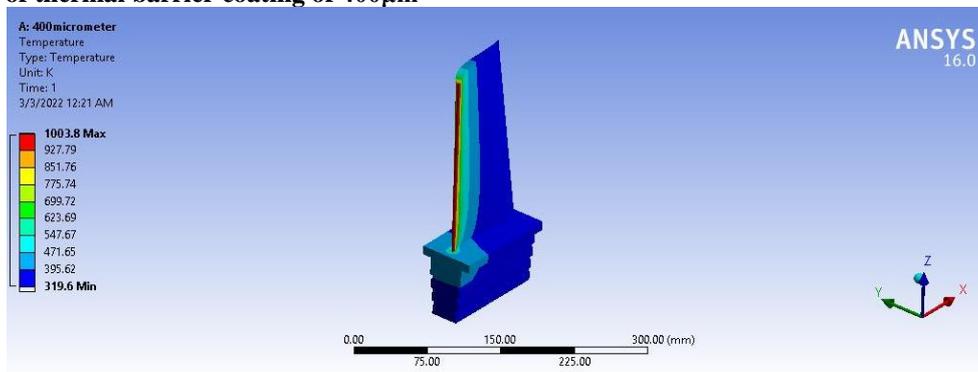


Figure 5.10.a. Temperature distribution

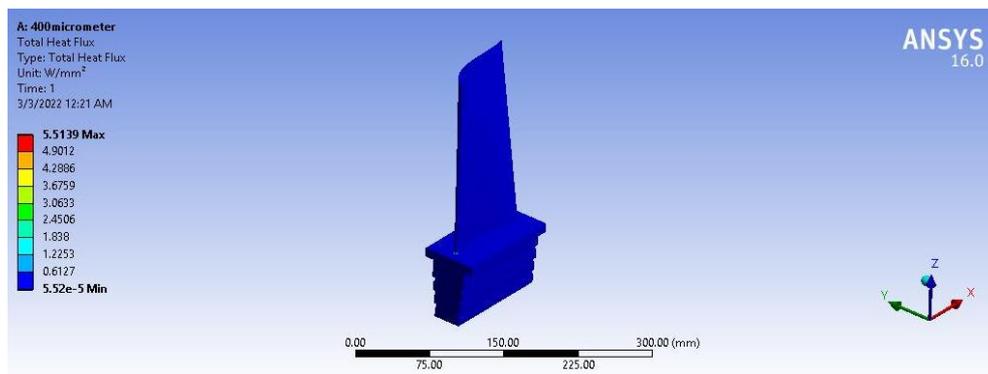


Figure 5.10.b. Heat flux experienced by the turbine blade structure

5.5.5. Results of thermal barrier coating of 500 μ m

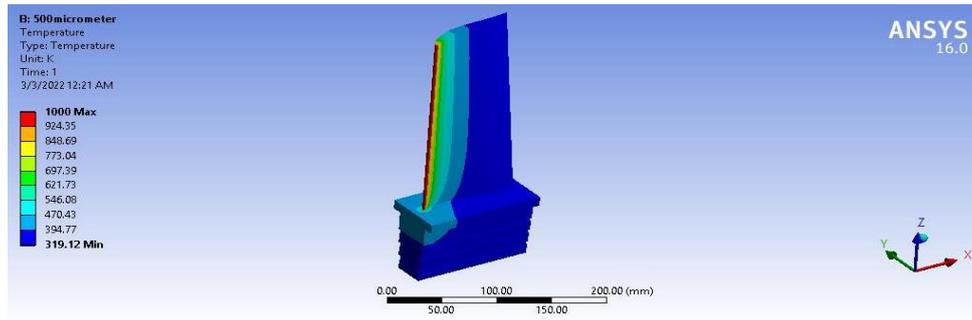


Figure 5.11.a. Temperature distribution

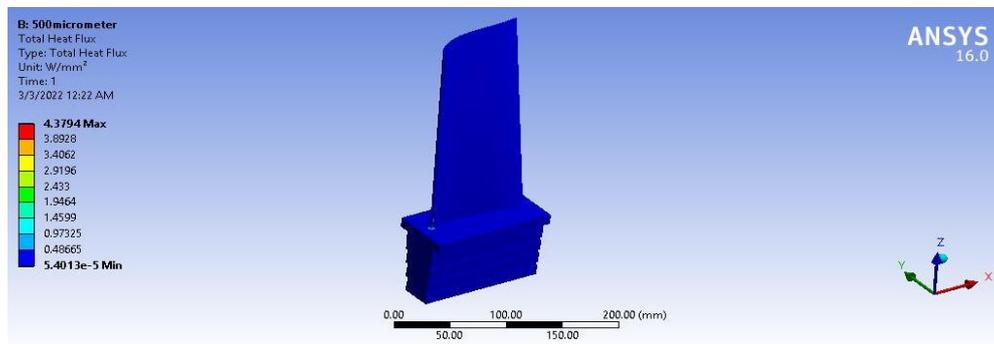


Figure 5.11.b. Heat flux experienced by the turbine blade structure

TABULATED RESULT RESULTS FOR VARIOUS COATING THICKNESS

Coating thickness (μm)	Temperature (K)		Heat flux (W/mm ²)	
	Min	Max	Min	Max
100	319.81	1000	1.81E-5	12.236
200	320.07	1000	3.89E-5	9.460
300	319.66	1000	1.43E-4	4.003
400	319.60	1000	5.52E-5	5.513
500	319.12	1000	5.40E-5	4.370

Table 5.6. Tabulated results for various coating thickness

CONCLUSION

A 3D numerical model of turbine blade and TBC with different thicknesses is developed in order to evaluate the protection rule of TBC thickness. The effect of temperature on material properties have been considered and thermal boundary conditions are modeled as heat transfer coefficient. Keeping assembly dimensions unchanged in this study, it is concluded

that TBC thickness has significant influence on temperature distribution of blade body and consequently on its life. Following results have been obtained:

Results show that TBC acts as an excellent insulation layer on the blade core, which can decrease temperature of blade up to 100 °C when TBC thickness changes from 100 to 500 μm. The temperature drop is not constants in all the regions and increases when moving from trailing edge to leading edge along either pressure or suction side.

By increasing TBC thickness, low temperature regions on the blade surface extend against high temperature regions.

By using the results of this simulation, the critical regions of blade surface and TBC, which are likely to failure during turbine operation, can be predicted. In addition, in our next plan, the temperature distribution is used in order to develop a Thermo-Elasto-Plastic model to evaluate stress fields through the blade and TBC. This analysis is important in case of investigation of residual stress in blade as well as crack initiation and propagation in TBC under thermal shock or thermal fatigue.

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